# Characteristics of the rotor R4-06 for the O4 NCal system VIR-0860B-22 

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This note is a revised version of the previous release. The rotor balance was unstable when operating at frequencies above 20 Hz , the unbalance seemed to come from the machining of one of the faces of the rotor. The rotor face has been machined again at IPHC to reduce this unbalance and so the measurements of the thickness have been updated in this note as well as the counterweight.

## 1 Introduction

This note follows the same discussion made on the rotor R4-01 in VIR-0591C-22 and R4-05 in VIR-0859A-22. The drawings and figures of the rotor can be found in this first technical note.

The rotor has been engraved IPHC-R4-06 on one side and sandblasted on the other side.

## 2 Measurement method

To determine the geometry of the rotor we will use the same method as for R4-01 (see VIR-0591C-22). The thickness was measured using $24 * 2=48$ points (see fig. 1 ), the outer diameter was measured in $4 * 2=8$ points and the inner diameter using 4 points.


Figure 1: Outline of the faces of the rotor with the measurement points. Left figure is face up, right figure is face down. Sectors have been labelled $L$ for left sector and $R$ for right sector.

The tool used to measure the thickness and the outer diameter is a measuring column "Garant 44 5350_600 HC1" (see VIR-0160A-22) with a given precision of $1.8+\mathrm{L} / 600 \mu \mathrm{~m}$ ( L the measured length in mm). A vernier caliper "TESA-CAL IP67" with a precision of $20 \mu \mathrm{~m}$ was used to measure the inner diameter.
The measuring column was operated on a metrology table with a value range from 0 to $2 \mu \mathrm{~m}$. The rms of the 16 values is $0.9 \mu \mathrm{~m}$.

We measured the opening angles of the sectors using a video measuring microscope "Garant MM2" (see VIR-0591C-22) with a given precision of $2.9+\mathrm{L} / 100 \mu \mathrm{~m}$ at $95 \%$ CL (L the measured length in mm ).

### 2.1 Thermal effects and density

The rotor R4-06 has been machined from the same aluminum block as R4-05 described in VIR-0859A-22. As for R4-05 the results will be expressed at a reference temperature of $23^{\circ} \mathrm{C}$.
The density of the rotor $\mathrm{R} 4-06$ is then $2810.8 \pm 0.2 \mathrm{~kg} . \mathrm{m}^{-3}$. This density is measured in air, if the rotor is used under vacuum, the density should be increased by the air density ( $\rho_{\text {air }}=1.3 \mathrm{~kg} \cdot \mathrm{~m}^{-3}$ ).

## 3 Raw measurements of the rotor

This section presents the raw measurements made on the rotor at the ambient temperature of $21^{\circ} \mathrm{C}$ for the thickness and $24.4^{\circ} \mathrm{C}$ for the diameter. Table 1 shows the thickness measurements according to the measurement points defined in figure 1. The rotor is laying on the table. The rotor surface as well as the table are not perfectly flat. Some space could be present in between that should be substracted when computing the rotor thickness as discussed later.

| Measurement point | L sector |  | Measurement point | R sector |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Up | Down |  | Up | Down |
| a | 104.331 | 104.308 | q | 104.341 | 104.343 |
| b | 104.332 | 104.322 | r | 104.343 | 104.348 |
| c | 104.338 | 104.336 | s | 104.347 | 104.353 |
| d | 104.345 | 104.341 | t | 104.351 | 104.362 |
| e | 104.332 | 104.337 | u | 104.327 | 104.318 |
| f | 104.336 | 104.340 | v | 104.326 | 104.331 |
| g | 104.340 | 104.344 | w | 104.333 | 104.343 |
| h | 104.345 | 104.354 | z | 104.339 | 104.351 |
| i | 101 | 483 | m | 101 | . 484 |
| j | 101 | 482 | n | 101 | 484 |
| k | 101 | 485 | o | 101 | 485 |
| 1 | 101 | 485 | p | 101 | 485 |

Table 1: Raw measurements of the height in mm for each point at $21^{\circ} \mathrm{C}$ on L and R sectors of $\mathrm{R} 4-06$.

Table 2 displays the diameter measurements. The measurements were made on $4 * 2$ diameters (two parts of each diameter, the up and down sides of the rotor).

| Measurement point | Up | Down |
| :---: | :---: | :---: |
| 1 | 207.942 | 207.939 |
| 2 | 207.973 | 207.967 |
| 3 | 207.976 | 207.980 |
| 4 | 207.933 | 207.960 |

Table 2: Raw measurements of the diameter in mm for each point at $24.4^{\circ} \mathrm{C}$ on $\mathrm{R} 4-06$.

Theoretical values were taken for the inner radius $r_{\min }=29 \mathrm{~mm}$ and the up face radius for the counterweight $r_{\text {counterweight }}=40 \mathrm{~mm}$ (see drawing in VIR-0591C-22).

## 4 Extracting the geometrical parameters

### 4.1 Thickness

We need to correct the possible gap between the rotor and the measuring table. Assuming that the table is flatter than the rotor surface we can extract the gap from the measurement of the top surface considering the plane tangents to the highest points (asking them to be on both sectors). For this rotor these points are $\mathrm{d}, \mathrm{s}, \mathrm{t}$ for the up and $\mathrm{h}, \mathrm{s}, \mathrm{t}$ for the down face (see fig. 1 ). Using the measurements in table 1 we can compute a plane equation for each side of the rotor in cartesian coordinates:

Up plane equation : $z=8.37 \times 10^{-5} x-2.34 \times 10^{-4} y+104.34$
Down plane equation : $z=8.51 \times 10^{-5} x-4.79 \times 10^{-4} y+104.34$
Using eqs. (1) and (2) the gap can be determined, see table 3. The maximum rms of the gap for each sector is $10 \mu \mathrm{~m}$.

| Measurement point | L sector |  | Measurement point | R sector |  |
| :---: | :---: | :---: | :---: | :---: | :---: |
|  | Up | Down |  | Up | Down |
| a | -9 | 5 | q | -5 | -11 |
| b | -4 | 5 | r | -1 | -6 |
| c | -2 | 8 | s | 0 | 0 |
| d | 0 | 19 | t | 0 | 0 |
| e | -3 | -13 | u | 7 | 8 |
| f | -3 | -7 | v | 17 | 11 |
| g | -2 | 0 | w | 18 | 16 |
| h | -2 | 0 | z | 18 | 22 |

Table 3: Gap computed in $\mu \mathrm{m}$ on up and down sides of both sectors of R4-06.

We can then compute the rotor thickness for each point by removing these gaps. If one of the raw values is lower than the corrected thickness we take this lowest value. The value of each point is shown in table 4 at $23^{\circ} \mathrm{C}$.

| Measurement point | L sector | Measurement point | R sector |
| :---: | :---: | :---: | :---: |
| a | 104.304 | q | 104.335 |
| b | 104.323 | r | 104.342 |
| c | 104.339 | s | 104.352 |
| d | 104.346 | t | 104.356 |
| e | 104.324 | u | 104.323 |
| f | 104.334 | v | 104.331 |
| g | 104.344 | w | 104.338 |
| h | 104.350 | z | 104.344 |
| i | 101.488 | m | 101.489 |
| j | 101.487 | n | 101.489 |
| k | 101.490 | o | 101.490 |
| l | 101.490 | p | 101.493 |

Table 4: Measurements of the thickness in mm for each point at $23^{\circ} \mathrm{C}$ on L and R sectors of $\mathrm{R} 4-06$.

### 4.2 Radius

Using comparators while the rotor is rotating on its axis we can determine the deformation on both sectors and compute different radii values. Table 5 shows the raw measurements using comparators on L and R sectors. The measurements were made on the up, center and down sides of $L$ and $R$ sectors using three comparators for a total of $5 * 3 * 2=30$ points (the first and last points are near the edge of the sectors).

| Measurement point | L sector |  |  | R sector |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Up | Center | Down | Up | Center | Down |
| A | -10 | 50 | 0 | -5 | 5 | -5 |
| B | 35 | 45 | 35 | 30 | 45 | 35 |
| C | 40 | 50 | 40 | 35 | 45 | 40 |
| D | 30 | 45 | 30 | 20 | 30 | 30 |
| E | 0 | 0 | 0 | -25 | -5 | -5 |

Table 5: Raw measurements in $\mu \mathrm{m}$ of the comparators for the $L$ and $R$ sectors of R4-06.

The zeroing of the comparators was made arbitrarily close to the edge of the sector. The offsets shown in table 5 are measured relative to this reference.

To compute the radius per measurement point we use the following process: First we compute the mean deformation for one comparator. Then we remove this mean deformation to each measurement of this comparator. The corrected shift value is added to the mean radius of 103.976 mm computed using table 2 at $23^{\circ} \mathrm{C}$. This process is repeated for each comparator. The final radius for each point are shown in table 6 .

| Measurement point | L sector |  |  | R sector |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Up | Center | Down | Up | Center | Down |
| A | 103.954 | 103.998 | 103.959 | 103.959 | 103.953 | 103.954 |
| B | 103.999 | 103.993 | 103.994 | 103.994 | 103.993 | 103.994 |
| C | 104.004 | 103.998 | 103.999 | 103.999 | 103.993 | 103.999 |
| D | 103.994 | 103.993 | 103.989 | 103.984 | 103.978 | 103.989 |
| E | 103.964 | 103.948 | 103.959 | 103.939 | 103.943 | 103.954 |

Table 6: Radius measurements in mm at $24.4^{\circ} \mathrm{C}$ for the L and R sectors of R4-06.

## 5 Characterization of the rotor using a simple model

### 5.1 Theoretical model of the rotor

As explained in VIR-0591C-22 the theoretical strain of the mirror at 2 f using FROMAGE v1r2 with the rotor design parameters:

$$
\operatorname{strain}(2 \mathrm{f})=\frac{2.1217 \times 10^{-18}}{\left(2 f_{r o t}\right)^{2}}
$$

This strain value will be compared to models based on the measurements of the rotor.

### 5.2 Thickness

A simple model can be used to determine a mean value for the thickness and its uncertainty.
As shown on fig. 1, a total of 16 points were used to compute the thickness of each sector. In this case we will not consider the inner points so that we obtain uniform sectors.

For the simple model we take the thickness as the mean value of table 4: 104.337 mm at $23^{\circ} \mathrm{C}$. Since we have a limited number of measurement points, to be conservative we take the thickness uncertainty as the rms of table $4(13.3 \mu \mathrm{~m})$ to which we add linearly the metrology table uncertainty $(0.9 \mu \mathrm{~m})$ and the tool uncertainty $(2.0 \mu \mathrm{~m})$. Therefore, for this simple model, the thickness is $104.377 \pm 0.016 \mathrm{~mm}$.

### 5.3 Radius

For the simple model we take the radius as the mean value of table 6: 103.976 mm at $23^{\circ} \mathrm{C}$. Using a linear sum of the rms of table $6(20.8 \mu \mathrm{~m})$ and the tool uncertainty ( $2.2 \mu \mathrm{~m}$ ) we take an uncertainty of $23 \mu \mathrm{~m}$ on the mean radius.
We have to point out that we do not take into account the fact that the sectors might not be centered on the same axis. Therefore the uncertainty might be underestimated. We will then consider each sector individually later.

### 5.4 Expected NCal signal and uncertainties

The geometry used to describe the rotor as a simple model is represented in figure 2.


Figure 2: Simple model geometry used to describe the rotor. Left is a front view, right is a side view.

Using the analytical equation of the strain at 2 f (see eq. 8 in Newtonian calibrator tests during the Virgo 03 data taking) we compute with our parameters ( $d=1.7 \mathrm{~m}$ and an angle $\phi=34.7^{\circ}$ ):
$\operatorname{strain}(2 \mathrm{f})=\frac{2.1185 \times 10^{-18}}{\left(2 f_{\text {rot }}\right)^{2}}$.
Using FROMAGE on this geometry we compute the following 2 f strain on the mirror at a distance of 1.7 m and an angle of $34.7^{\circ}$ :
$\operatorname{strain}(2 f)=\frac{2.1186 \times 10^{-18}}{\left(2 f_{\text {rot }}\right)^{2}}$
Comparing the theoretical model strain with the simple model at 2 f using FROMAGE we obtain a relative deviation of $0.146 \%$.
Comparing the analytical strain at 2 f with FROMAGE we obtain a relative deviation of $0.005 \%$.
The uncertainties considered for this model are displayed in table 7.

| R4-06 rotor parameter simple model $\left(23^{\circ} \mathrm{C}\right)$ |  |  | NCal 2f signal uncertainty |  |
| :---: | :---: | :---: | :---: | :---: |
| name | value | uncertainty | formula | value (\%) |
| Density $\rho\left(\mathrm{kg} \cdot \mathrm{m}^{-3}\right)$ | 2810.8 | 0.2 | $\delta \rho / \rho$ | 0.007 |
| Thickness $b(\mathrm{~mm})$ | 104.337 | $1.6 \times 10^{-2}$ | $\delta b / b$ | 0.015 |
| $r_{\max }(\mathrm{mm})$ | 103.976 | $2.3 \times 10^{-2}$ | $4 \delta r_{\max } / r_{\max }$ | 0.088 |
| $G\left(\mathrm{~m}^{3} \cdot \mathrm{~kg}^{-1} \cdot \mathrm{~s}^{-2}\right)$ | $6.67430 \times 10^{-11}$ | $1.5 \times 10^{-15}$ | $\delta G / G$ | 0.002 |
| Temperature $T\left({ }^{\circ} \mathrm{C}\right)$ | 23 | 3 | $\left\|\frac{\partial h}{\partial T}\right\| \frac{\Delta T}{h}$ | 0.014 |
| Quadratic sum |  |  |  |  |

Table 7: Uncertainties on the amplitude of the calibration signal at 2 f from the R4-06 rotor simple model geometry.

## 6 Characterization of the rotor using an advanced model

### 6.1 Thickness

A more advanced model can be used considering the deformations on the surfaces of the sectors for better accuracy. Each measurement point of table 4 can be considered as a sub-sector with its own thickness.
The uncertainty on this value is more complex to evaluate. As a conservative approach we use the maximum rms of the deviation to a plane for each sector ( $10 \mu \mathrm{~m}$ see section 4.1 ) to which we add linearly the uncertainty on the flatness of the measurement table $(0.9 \mu \mathrm{~m})$ as well as the measurement tool $(2.0 \mu \mathrm{~m})$. The total uncertainty on the thickness is $13 \mu \mathrm{~m}$.

### 6.2 Radius

On fig. 1 we divided the external sectors in 4 sub-sectors for each sector (blue points). We convert the point of table 6 to the grid of fig. 1 by averaging the two closest values and converting them to $23^{\circ} \mathrm{C}$. The results are shown in table 8 . We notice that the L sector is on average $7 \mu \mathrm{~m}$ larger than the R sector.

| Radius | L sector |  |  | R sector |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  | Up | Center | Down | Up | Center | Down |
| 1 | 103.973 | 103.992 | 103.973 | 103.973 | 103.970 | 103.971 |
| 2 | 103.998 | 103.992 | 103.993 | 103.993 | 103.990 | 103.993 |
| 3 | 103.996 | 103.992 | 103.991 | 103.988 | 103.982 | 103.991 |
| 4 | 103.976 | 103.967 | 103.971 | 103.958 | 103.957 | 103.968 |

Table 8: Radius measurements (in mm at $23^{\circ} \mathrm{C}$ ) for the L and R sectors of $\mathrm{R} 4-06$.

The maximum rms of the radii for each sector is $13.3 \mu \mathrm{~m}$. The tool uncertainty is $2.2 \mu \mathrm{~m}$. Like for the thickness we use a linear sum and find the uncertainty on both radii to be $16 \mu \mathrm{~m}$.

### 6.3 Counterweight

To balance this rotor, two masses $m_{1}$ and $m_{2}$ were respectively placed at angles $\alpha_{1}$ and $\alpha_{2}$ and at a radius $r_{m}$. Using the following formula we can compute a single mass equivalent to the moment of the two masses:

$$
\begin{equation*}
m=\sqrt{m_{1}^{2}+m_{2}^{2}+2 m_{1} m_{2} \cos \left(\alpha_{1}-\alpha_{2}\right)} \tag{3}
\end{equation*}
$$

Using the result of eq. (3) and the following formulas (where $\rho_{\mathrm{Al}}$ is the density of the aluminum counterweight, $h$ its thickness and $R$ its outer radius) we can compute the angle of the chord cut ( $\gamma_{m}$ ) on the counterweight and its position angle ( $\alpha_{m}$ ):

$$
\begin{gather*}
\gamma_{m}=\arcsin \left[\left(\frac{3}{2} \frac{m r_{m}}{\rho_{\mathrm{Al}} h R^{3}}\right)^{1 / 3}\right]  \tag{4}\\
\alpha_{m}=\arctan \frac{m_{1} \sin \left(\alpha_{1}\right)+m_{2} \sin \left(\alpha_{2}\right)}{m_{1} \cos \left(\alpha_{1}\right)+m_{2} \cos \left(\alpha_{2}\right)} \tag{5}
\end{gather*}
$$

A counterweight has been designed to reduce the unbalance of the rotor. This counterweight is made of aluminum $2017\left(\rho_{\mathrm{Al}_{2017}}=2790 \mathrm{~kg} . \mathrm{m}^{-3}\right)$ and the geometry is shown in fig. 3.

The dimensions of the counterweight are: an inner radius of 10 mm , an outer radius of 40 mm and a material cut with a chord of 70.36 mm to balance the rotor (see hatched are on fig. 3). Three screws were placed on holes 1, 3 and 4.


Figure 3: Outline of the counterweight for R4-06. The hatched area represents the material removed.

### 6.4 Opening angles and asymmetry

The opening angles of the full and empty sectors have been measured using a video microscope with the same method as for R4-01. The uncertainty on $\alpha=0.2 \mathrm{mrad}$ is the same as for R4-01 giving an uncertainty of $4 \times 10^{-6} \%$ on the 2 f signal which is neglectable.

The measurements are shown in table 9. The center value corresponds to the mean of up and down measurements.

| Opening angle | Up | Center | Down |
| :---: | :---: | :---: | :---: |
| L | 1.57072 | 1.57091 | 1.57109 |
| R | 1.57092 | 1.57091 | 1.57091 |
| L-R | 1.57088 | 1.57085 | 1.57083 |
| R-L | 1.57066 | 1.57051 | 1.57036 |

Table 9: Opening angle measurements in rad for the $\mathrm{L}, \mathrm{R}$ full sectors and L-R, R-L empty sectors of R4-06.

These measurements allow us to compute the signal with different opening angles and an asymmetry between the sectors. These measured opening angles will be included in the advanced model described in the next section.

### 6.5 Expected NCal signals and uncertainties

### 6.5.1 Advanced geometry including chamfers and counterweight

The geometry used to describe the rotor as an advanced model is represented in figure 4. The external parts of the sectors are divided in 3 sub-sectors each to correspond to the different radii determined. In addition we include the counterweight, the screws, the screw holes, the opening angles and asymmetry of the sectors.


Figure 4: Advanced model geometry used to describe the rotor. Top left is a front view, top right is a side view (external sub-sectors) and bottom is a tilted view of the sectors. Only the 4 external part sectors are divided in 3 sub-sectors each. The chamfers are visible on the inner radius.

Using FROMAGE on this geometry gives the following strains:

- $\operatorname{strain}(1 \mathrm{f})=\frac{5.3509 \times 10^{-20}}{\left(1 f_{\text {rot }}\right)^{2}}$
- $\operatorname{strain}(2 \mathrm{f})=\frac{2.1187 \times 10^{-18}}{\left(2 f_{\text {rot }}\right)^{2}}$
- $\operatorname{strain}(3 f)=\frac{7.6454 \times 10^{-24}}{\left(3 f_{r o t}\right)^{2}}$

The relative deviation to the simple model at 2 f is $0.005 \%$.

### 6.5.2 Remaining geometry uncertainty

Since R4-02 and following have been machined with a better precision than R4-01 as seen from the lower 1f value. The remaining geometry uncertainties are taking as for R4-01.

### 6.5.3 Uncertainties

To set an uncertainty on the strain(2f) from the description of the geometry we take the difference between the simple model $\left(\operatorname{strain}(2 f)=2.1186 \times 10^{-18} /(2 f)^{2}\right)$ and the advanced model $(\operatorname{strain}(2 f)=2.1187 \times$ $10^{-18} /(2 f)^{2}$ ). This deviation, $0.005 \%$, is reported in table 10 as modelling uncertainty.

The uncertainties considered for this full model are displayed in table 10.

| R4-06 rotor parameter advanced model ( $23^{\circ} \mathrm{C}$ ) |  |  | NCal 2f signal uncertainty |  |
| :---: | :---: | :---: | :---: | :---: |
| name | mean value | uncertainty | formula | value (\%) |
| Density $\rho$ (kg.m ${ }^{-3}$ ) | 2810.8 | 0.2 | $\delta \rho / \rho$ | 0.007 |
| Thickness $b$ left sector (12 sub-sectors) (mm) Thickness $b$ right sector ( 12 sub-sectors) (mm) | $\begin{aligned} & 104.333 \\ & 104.340 \end{aligned}$ | $1.3 \times 10^{-2}$ | $\delta b / b$ | 0.012 |
| $r_{\text {max }}$ left sector (12 ext sub-sectors) (mm) $r_{\text {max }}$ right sector (12 ext sub-sectors) (mm) | $\begin{aligned} & 103.985 \\ & 103.978 \end{aligned}$ | $1.4 \times 10^{-2}$ | $4 \delta r_{\text {max }} / r_{\text {max }}$ | 0.059 |
| $G\left(\mathrm{~m}^{3} \cdot \mathrm{~kg}^{-1} . \mathrm{s}^{-2}\right)$ | $6.67430 \times 10^{-11}$ | $1.5 \times 10^{-15}$ | $\delta G / G$ | 0.002 |
| Temperature $T\left({ }^{\circ} \mathrm{C}\right)$ | 23 | 3 | $\left.\frac{\partial h}{\partial T} \right\rvert\, \frac{\Delta T}{h}$ | 0.014 |
| Modelling Uncertainty |  |  |  | 0.005 |
| FROMAGE grid uncertainty |  |  |  | 0.005 |
| Opening angle and sector asymmetry uncertainty |  |  |  | $<5 \times 10^{-4}$ |
| Remaining geometry uncertainty |  |  |  | $<5 \times 10^{-4}$ |
| Total uncertainty from the rotor (quadratic sum) |  |  |  | 0.063 |

Table 10: Uncertainties on the amplitude of the calibration signal at 2 f from the R4-06 rotor advanced model geometry at $23^{\circ} \mathrm{C}$.

## A Appendix

```
### This is a cfg file for a more realistic geometry of the mirror and the Virgo NCal R4-06 (2022)
### ALL THE OBJECTS ARE DEFINED IN THE MIRROR'S FRAME (0,x,y,z),
### with O the center of the mirror, x axis along the ITF's beam toward the beam-splitter,
### y axis orthogonal to x in the plane of the ITF,
### z axis orthogonal to the plane of the ITF upward
### MIRROR DEFINITION
```

GRID_SIZE 12308
CYLINDER 2202. 00.1750 .2360000
GRID_SIZE 111
\# Defining the flats on the edge of the mirror
CUT_CYL 2202. 0.1750 .20 .0500
CUT_CYL 2202. 0.1750 .20 .050180
\# Defining the ears and anchors of the mirror
CUBOID 2202. $0.0900 .0100 .015000 .1782-0.0125$
CUBOID 2202. $0.0900 .0100 .015 \quad 0 \quad-0.1782-0.0125$
CUBOID 2202. $0.0390 .008 \quad 0.008-0.02-0.1772-0.024$
CUBOID 2202. $0.039 \quad 0.008 \quad 0.008-0.02 \quad 0.1772-0.024$
CUBOID 2202. $0.0390 .0080 .008 \quad 0.02-0.1772-0.024$
CUBOID 2202. $0.0390 .008 \quad 0.008 \quad 0.02 \quad 0.1772-0.024$
\#\#\# ROTOR DEFINITION: CYLINDER DENSITY INNER_RADIUS OUTER_RADIUS THICKNESS OPEN_ANGLE r z theta
ROTOR_CYLINDRICAL 1.734 .700
\#\#\# COUNTERWEIGHT 2790.
GRID_SIZE 166540
CYLINDER 2790. 0.0100 .0400 .00336000 .0495130
GRID_SIZE 111
CUT_CYL 2790. 0.0400 .0030 .07036075 .76
\#\#\# SCREW HOLES
GRID_SIZE 444
CYLINDER -2810.8 00.00150 .0123600 .030 .0447438949988150
CYLINDER -2810.8 00.00150 .0123600 .030 .04474489504630
CYLINDER -2810.8 00.00150 .0123600 .030 .0447443950224210
CYLINDER -2810.8 00.00150 .0123600 .030 .0447463951168330
\#\#\# SCREWS COUNTERWEIGHT
CYLINDER 4810. 00.00150 .0033600 .030 .0493929708353150
CYLINDER 7600. 00.00150 .006653600 .030 .0440963951168150
CYLINDER 4810. 00.00150 .0033600 .030 .0493929708353210
CYLINDER 7600. 00.00150 .006653600 .030 .0440963951168210
CYLINDER 4810. 00.00150 .0033600 .030 .0493929708353330
CYLINDER 7600. 00.00150 .006653600 .030 .0440963951168330

```
# TRES RAPIDE
#GRID_SIZE 4 4 4
# RAPIDE
GRID_SIZE 8 17 14
# LENT
#GRID_SIZE 8 65 40
### L sector
## Inner part
OUTER_FILLET 2810.8 0.029 0.101488 -0.002850 0.01 -11.25 146.2477
CYLINDER 2810.8 0.029 0.04 0.101488 22.5 0 -0.002850 146.2477
CYLINDER 2810.8 0.029 0.04 0.101487 22.5 0 -0.002851 168.7492
CYLINDER 2810.8 0.029 0.04 0.101490 22.5 0 -0.002850 191.2508
CYLINDER 2810.8 0.029 0.04 0.101490 22.5 0 -0.002850 213.7523
OUTER_FILLET 2810.8 0.029 0.101490 -0.002850 0.01 11.25 213.7523
## Middle part
CYLINDER 2810.8 0.04 0.071998 0.104324 22.5 0 0 146.2477
CYLINDER 2810.8 0.04 0.071998 0.104334 22.5 0 0 168.7492
CYLINDER 2810.8 0.04 0.071998 0.104344 22.5 0 0 191.2508
CYLINDER 2810.8 0.04 0.071998 0.104350 22.5 0 0 213.7523
## Outer part
CYLINDER 2810.8 0.071998 0.103973 0.03476802565452 22.5 0 0.03476802565452 146.2516
CYLINDER 2810.8 0.071998 0.103992 0.03476802565452 22.5 0 0 146.2477
CYLINDER 2810.8 0.071998 0.103973 0.03476802565452 22.5 0 -0.03476802565452 146.2438
CYLINDER 2810.8 0.071998 0.103998 0.0347744625426994 22.5 0 0.0347744625426994 168.7505
CYLINDER 2810.8 0.071998 0.103992 0.0347744625426994 22.5 0 0 168.7492
CYLINDER 2810.8 0.071998 0.103993 0.0347744625426994 22.5 0 -0.0347744625426994 168.7479
CYLINDER 2810.8 0.071998 0.103996 0.03477980355084 22.5 0 0.03477980355084 191.2495
CYLINDER 2810.8 0.071998 0.103992 0.03477980355084 22.5 0 0 191.2508
CYLINDER 2810.8 0.071998 0.103991 0.03477980355084 22.5 0 -0.03477980355084 191.2521
CYLINDER 2810.8 0.071998 0.103976 0.03478197387041 22.5 0 0.03478197387041 213.7484
CYLINDER 2810.8 0.071998 0.103967 0.03478197387041 22.5 0 0 213.7523
CYLINDER 2810.8 0.071998 0.103971 0.03478197387041 22.5 0 -0.03478197387041 213.7562
### R sector
## Inner part
OUTER_FILLET 2810.8 0.029 0.101489 -0.002857 0.01 11.25 33.7427
CYLINDER 2810.8 0.029 0.04 0.101489 22.5 0 -0.002857 33.7427
CYLINDER 2810.8 0.029 0.04 0.101489 22.5 0 -0.002857 11.2410
CYLINDER 2810.8 0.029 0.04 0.101490 22.5 0 -0.002856 348.7394
CYLINDER 2810.8 0.029 0.04 0.101493 22.5 0 -0.002853 326.2377
OUTER_FILLET 2810.8 0.029 0.101493 -0.002853 0.01 -11.25 326.2377
## Middle part
CYLINDER 2810.8 0.04 0.071998 0.104335 22.5 0 0 33.7427
CYLINDER 2810.8 0.04 0.071998 0.104342 22.5 0 0 11.2410
CYLINDER 2810.8 0.04 0.071998 0.104352 22.5 0 0 348.7394
CYLINDER 2810.8 0.04 0.071998 0.104356 22.5 0 0 326.2377
```

```
## Outer part
CYLINDER 2810.8 0.071998 0.103973 0.03477430793653 22.5 0 0.03477430793653 33.7428
CYLINDER 2810.8 0.071998 0.103970 0.03477430793653 22.5 0 0 33.7427
CYLINDER 2810.8 0.071998 0.103971 0.03477430793653 22.5 0-0.03477430793653 33.7426
CYLINDER 2810.8 0.071998 0.103993 0.03477697472907 22.5 0 0.03477697472907 11.2411
CYLINDER 2810.8 0.071998 0.103990 0.03477697472907 22.5 0 0 11.2410
CYLINDER 2810.8 0.071998 0.103993 0.03477697472907 22.5 0 -0.03477697472907 11.2410
CYLINDER 2810.8 0.071998 0.103988 0.03477930817253 22.5 0 0.03477930817253 348.7393
CYLINDER 2810.8 0.071998 0.103982 0.03477930817253 22.5 0 0 348.7394
CYLINDER 2810.8 0.071998 0.103991 0.03477930817253 22.5 0 -0.03477930817253 348.7394
CYLINDER 2810.8 0.071998 0.103958 0.03478130826693 22.5 0 0.03478130826693 326.2376
CYLINDER 2810.8 0.071998 0.103957 0.03478130826693 22.5 0 0 326.2377
CYLINDER 2810.8 0.071998 0.103968 0.03478130826693 22.5 0 -0.03478130826693 326.2378
### GENERAL PARAMETERS
STEP 22.5 16
ARM_LENGTH 3000
SIGNAL 3
```

